### Development of Co-Firing Technology of Pulverized Coal and Ammonia for Suppressing NO<sub>x</sub> Generation

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In order to reduce  $CO_2$  emissions from coal-fired boilers, there are expectations of ammonia, which is free of carbon content, if it is used as fuel. On the other hand, there is a concern that the NO<sub>x</sub> emissions will increase because ammonia has a higher nitrogen content than coal. In this study, using a 10 MWth test furnace, an ammonia and pulverized coal co-firing test was conducted with ammonia injection velocity as a parameter at 20% co-firing ratio. The result showed that NO<sub>x</sub> emissions of co-firing are the same as those of single coal combustion. In addition, a co-firing test for evaluating the effect of two-stage combustion ratio, heat input and fuel ratio on NO<sub>x</sub> was conducted. The results clarified the conditions for reducing NO<sub>x</sub> emissions in ammonia co-firing.

### 1. Introduction

Amid a need to reduce  $CO_2$  emissions from coal-fired boilers in an effort to prevent global warming, there are expectations of the utilization of hydrogen as a fuel free of carbon content. However, the cost of transportation and storage is one factor inhibiting the popularization of hydrogen. As a solution, ammonia is attracting much attention as an energy carrier with a high hydrogen content that is relatively easy to liquefy, transport, and store compared to hydrogen. However, since ammonia contains a large amount of nitrogen, there is concern that if it is used as fuel for coal-fired boilers, the amount of  $NO_x$  in flue gas will increase compared with conventional single coal combustion.

The NO<sub>x</sub> produced from coal-fired boilers is mainly fuel NO<sub>x</sub> produced by oxidation of nitrogen contained in coal than thermal NO<sub>x</sub> generated by oxidation of nitrogen in air. Various measures have been implemented to reduce NO<sub>x</sub> emissions, such as the two-stage combustion which divides combustion into two stages, whereby fuel-rich combustion is performed in the first stage and NO<sub>x</sub> generation is suppressed, then complete combustion is performed in the second stage. In two-stage combustion, by performing firststage combustion at a relatively low air ratio, discharge of the volatile matter within pulverized coal is promoted at the same time as achieving formation of reducing flame, therefore resulting in suppression of  $NO_x$  generation<sup>(3)</sup>. There was concern that it may not be possible to suppress NO<sub>x</sub> generation using the two-stage combustion method described above when nitrogen-rich ammonia and pulverized coal are co-fired.

In this study, we conducted a pulverized coal/ammonia cofiring test using a combustion test furnace with a heat input of 10 MWth class at a co-firing ratio of 20% (lower heating value (LHV) base) and succeeded in achieving stable combustion while suppressing NO<sub>x</sub> generation to the same extent as single coal combustion. Moreover, the effect of two-stage combustion ratio on the flue gas characteristics during ammonia co-firing was investigated. Therefore, setting appropriate two-stage combustion ratio enables ammonia co-firing with the same NO<sub>x</sub> emission level as single coal combustion. This suggests the possibility that ammonia reduces NOx generated in the first-stage combustion region. Furthermore, in order to apply ammonia co-firing technology to actual boiler furnaces, it is necessary to clarify the feasibility of both co-firing under a low-load condition and co-firing various coal types. For this purpose, co-firing tests with different heat input and coal fuel ratio were carried out, and the effect on the flue gas composition was investigated. These tests confirmed that NO<sub>x</sub> emissions of ammonia co-firing are the same as those of single coal combustion even under a low load condition, which suggests that ammonia co-firing at partial load is possible when applied to commercial furnaces. In addition, as a result of ammonia co-firing under the same combustion conditions using coal with different fuel ratio, it was confirmed that there was no significant difference in the NO<sub>x</sub> emissions.

#### 2. Test equipment and methods

#### 2.1 Test equipment

The pulverized coal/ammonia co-firing test was performed using a combustion test furnace with a heat input of 10 MWth class. Figure 1 shows the main flow diagram of the combustion test furnace. A burner is positioned on the front of the combustion test furnace. The air supplied from the forced draft fan is heated with the flue gas in the air heater and transported to the burner as combustion air. Part of the combustion air is supplied to the over air port installed at the top of the burner. The air supplied by the mill air fan is also heated with flue gas in the air heater then fed to the pulverizer to be used for drying the coal and transporting the pulverized coal. The pulverized coal is stored in a pulverized coal bin. A predetermined amount of the stored pulverized coal is ejected from the pulverized coal bin with the pulverized coal feeder and fed to the burner with the conveyance air supplied from the primary air fan. The induced draft fan draws in air supplied to the furnace to create negative pressure inside the furnace. The flue gas is sampled at the exit of the test furnace and its composition is measured by a gas analyzer.

Figure 2 is a schematic of the burner. The burner exit is axisymmetric and the burner has an inner cylinder, outer cylinder, air register, and wind box from the center. Pulverized coal discharged from the pulverized coal bin is supplied with the primary air along the flow path between the inner and outer cylinders. The combustion air is supplied through the outside of the outer cylinder and along a path surrounded by the wind box. The air register positioned in the combustion air flow path has multiple movable vanes in axisymmetric positions, and the combustion air's swirl force can be adjusted by changing the angles of these vanes. The recirculation flow is established in the front of the burner when the combustion air is swirled. When the hightemperature combustion gas and fuel mix rapidly due to the recirculation flow, discharge of the volatile matter and nitrogen-containing substance within the fuel is promoted, and this achieves both the formation of a stable flame and reduction of  $NO_x$  in the reducing atmosphere.

The ammonia supply unit feeds ammonia to the burner and is capable of continuously supplying up to 0.38 ton/h of ammonia. Four cylinders with 0.5 ton capacities can be installed on the ammonia supply unit which is also equipped with two hot water evaporators. Ammonia is vaporized by passing through the evaporators and is transported separately



Fig. 2 Schematic of burner

without mixing with air to a special-purpose nozzle installed in the center of the pulverized coal burner while controlling the mass flow rate. When ammonia is supplied, the pulverized coal is combusted in advance and the ammonia is blown in towards the flame.

### 2.2 Test method

**Table 1** shows the properties of the coal and ammonia used in the test. Two types of coal with different fuel ratios were prepared. The heat input of the combustion tests is set to between 7 and 10 MWth. The flow rate of ammonia is adjusted so that the co-firing rate is 20% in this series of tests. In order to evaluate the difference of NO<sub>x</sub> in flue gas during ammonia co-firing from that during single coal combustion, flue gas is continuously sucked at the test furnace exit and NO<sub>x</sub> is measured with an analyzer. In addition, flue gas is sucked and sampled with a probe and N<sub>2</sub>O concentration, NH<sub>3</sub> concentration and unburnt coal (UBC) in the ash were analyzed.

### 3. Test results and insights

# 3.1 Effect of ammonia injection velocity on NO<sub>x</sub> emissions and feasibility of pulverized coal/ ammonia co-firing

**Figure 3** shows time series data of  $CO_2$  and  $O_2$  concentration in the flue gas and NH<sub>3</sub> concentration at the inlet of the stack for single coal combustion and ammonia co-firing. Ammonia co-firing was performed under the ammonia injection velocity conditions mentioned hereinafter. It was confirmed



Fig. 1 Main flow diagram of test furnace

Item		Unit	Basis	Coal A	Coal B	Ammonia
Lower heating value		MJ/kg	Dry	29.0	29.1	18.5
Proximate analysis	Moisture	wt%	Air dried	2.2	2.0	_
	Fixed carbon	wt%	Dry	56.2	48.1	_
	Volatile matter	wt%	Dry	33.0	41.6	_
	Ash	wt%	Dry	10.8	10.3	_
	Fuel ratio (FR)	wt%	Dry	1.70	1.16	_
Ultimate analysis	Carbon	wt%	Dry	71.1	69.7	_
	Hydrogen	wt%	Dry	4.6	5.3	_
	Nitrogen (FN)	wt%	Dry	1.4	1.1	_
	Oxygen	wt%	Dry	11.7	13.2	_
	Sulfur	wt%	Dry	0.4	0.5	

Table 1 Properties of coal and ammonia



Fig. 3 Time history of flue gas composition. In both cases of single coal combustion and ammonia co-firing, CO<sub>2</sub>, O<sub>2</sub> and NH<sub>3</sub> concentration at inlet of stack have small fluctuation in time.

that at ammonia co-firing, compared to single coal combustion, the amount of coal combustion reduced by 20% and  $CO_2$  concentration in the flue gas also reduced by 20%. NH<sub>3</sub> concentration was equivalent to that of single coal combustion, and it is considered that the injected ammonia is practically completely burned in the furnace. The small fluctuations in gas composition show stable combustion.

**Figure 4** shows the burning flame at single coal combustion and 20% ammonia co-firing. **Figure 4-(b)** shows the flame created by ammonia co-firing when the swirl force of the combustion air is relatively weak. We confirmed that, in this case, the ignition point of the flame is further from the burner port than that of single coal combustion. This is considered to be due to the ammonia with low burning velocity.



Fig. 4 Flame appearance. In case of ammonia co-firing with weak swirl of combustion air, ignition point is distant from burner port compared with that of single coal combustion. With strong swirl of combustion air, it is the same level as that of single coal combustion

**Figure 4-(c)** shows the flame in ammonia co-firing where the swirl force of the combustion air has been increased and adjusted. At this condition, the ignition point of the flame is similar to that of single coal combustion.

Due to the results of stable flue gas composition over time, absence of unburned ammonia, and formation of a flame similar to that of single coal combustion, we judge ammonia co-firing to be possible using the method of ammonia injection from the center of the pulverized coal burner. The ignition point of the flame is adjusted to the same as that of single coal combustion by the ammonia injection velocity.

Below described is a pulverized coal/ammonia co-firing burner. The pulverized coal/ammonia co-firing burner developed for this study is of a structure whereby ammonia is blown into a recirculating flow area where the flame forms in the front of the burner. This structure aims to suppress oxidation of ammonia by supplying ammonia to a strong reduction zone. To investigate the effect of the ammonia injection velocity from the center of the burner to the furnace on  $NO_x$  emissions, the test was conducted for two ammonia velocities, a base velocity and a velocity five times the base flow velocity. The base velocity was determined by numerical simulation to have sufficient momentum to ensure the ammonia plunges into the recirculating flow of the pulverized coal flame.

**Figure 5** shows the relationship between the ammonia injection velocity,  $NO_x$  concentration and NO conversion ratio (*CR*) under the conditions of a heat input of 10 MWth, a two-stage combustion ratio of 30%, an air ratio of approx. 1.2, and an ammonia co-firing ratio of 20%. The ammonia injection velocity used in the **Fig. 5** is a normalized, relative value setting the base velocity as 1. *CR* is defined with formula (1).

$$CR = \frac{N_{NO_x}}{N_{fuel}} \qquad (1)$$

Here,  $N_{NO_x}$  expresses the nitrogen level in flue gas NO<sub>x</sub> and  $N_{fuel}$  expresses the nitrogen level contained in the injected fuel (coal and ammonia).

Under the condition of the co-firing with the base ammonia injection velocity,  $NO_x$  concentration was 144 ppm compared to 154 to 171 ppm at single coal combustion, which was 6 to 16% lower than that at single coal combustion. However, when the ammonia injection velocity five times the base was used,  $NO_x$  concentration was 187 ppm, which was between 20 and 30 ppm greater than that at single coal combustion. This is considered that because if the ammonia injection velocity is too high, the ammonia will not react in the burner recirculating flow zone and oxidation of the ammonia in the two-stage combustion region leads to an increase of  $NO_x$ .

The abovementioned result suggests that if the ammonia is supplied from the burner center at the appropriate velocity,  $NO_x$  can be suppressed to the same level as single coal combustion.

# 3.2 Effect of two-stage combustion ratio and air ratio on flue gas composition

Two-stage combustion ratio ( $R_{t,s}$ ), which is later defined by Equation (3), is an important parameter for controlling NO<sub>x</sub> emissions in coal-fired boilers. To investigate whether the adjustment of  $R_{t,s}$  enables to control NO<sub>x</sub> emissions in flue gas during ammonia co-firing, the combustion test with  $R_{t,s}$ as a parameter was conducted. Set values in the test are as follows: a heat input of 10 MWth, an air ratio of 1.1 to 1.2, and an ammonia co-firing ratio of 20%.

**Figures 6-(a)** and **-(b)** show the relationship between  $R_{ts}$  and NO<sub>x</sub> concentration in flue gas and the relationship between  $R_{ts}$  and CR, respectively. NO<sub>x</sub> concentration of single coal combustion decreases monotonically as  $R_{ts}$  increases from 20 to 40%. In contrast, NO<sub>x</sub> concentration of the ammonia co-firing decreases as  $R_{ts}$  increases between 20 and 30% and increases as  $R_{ts}$  increases over 30%. This is considered to be because UBC, which had remained until the two-stage combustion process, increased and burned with air for two-stage combustion with high  $R_{ts}$  at ammonia co-firing. In the test conditions, the *CR* was 4 to 10% at single coal combustion and 0.2 to 0.4% at ammonia co-firing. The total amount of nitrogen of the fuel injected into the furnace



Fig. 5  $NO_x$  concentration and *CR* as a function of relative velocity of ammonia. In the case of ammonia co-firing with base velocity of ammonia,  $NO_x$  concentration is 6 to 16% smaller than that of single coal combustion.



Fig. 6 NO<sub>x</sub> concentration and *CR* as a function of  $R_{t.s.}$  NO<sub>x</sub> concentration of the single coal combustion decreases monotonically as  $R_{t.s.}$  increases from 20 to 40%. In contrast, the NO<sub>x</sub> concentration of the ammonia co-firing has the minimum value at 30% of  $R_{t.s.}$ 

was approximately 17 times larger at ammonia co-firing than at single coal combustion.  $NO_x$  concentration is almost the same level, therefore the *CR* at ammonia co-firing is one-seventeenth compared to single coal combustion.

**Figure 7** shows the relationship between  $R_{t.s}$  and UBC in the ash. The amount of UBC in the ash was the same for both ammonia co-firing and single coal combustion. Therefore, by setting appropriate  $R_{t.s}$ , ammonia co-firing with the same NO<sub>x</sub> emission level as single coal combustion is possible.

Then, the effect of first-stage air ratio  $(\lambda_1)$  on NO is described below. In this test, air ratio  $(\lambda)$ ,  $R_{t,s}$ ,  $\lambda_1$ , and second-stage air ratio  $(\lambda_2)$  are defined by the following formulas.





Fig. 7 UBC as a function of  $R_{t.s.}$  In the condition  $R_{t.s}$  is from 20 to 40%, UBC of ammonia co-firing is on the same level as that of single coal combustion.

$$R_{t.s} = \frac{\text{Two-stage combustion air flow rate}}{\text{Full combustion air flow rate}} \times 100 \, [\%]$$

**Figure 8** shows the relationship between  $\lambda_1$  and *CR* when  $\lambda_2$  is between 0.3 and 0.4.

In the case of ammonia co-firing, when  $\lambda_1$  decreases from 0.9 to 0.8, the *CR* decreases, however this vicinity of 0.8 is the minimum point, and when  $\lambda_1$  decreases further, the *CR* begins to increase. For single coal combustion, however, *CR* decreases monotonically as  $\lambda_1$  decreases from 0.9 to 0.75.



Fig. 8 *CR* as a function of  $\lambda_1$ . In the case of ammonia co-firing, *CR* has the minimum point at about 0.8 of  $\lambda_1$ . In contrast, with single coal combustion, it monotonically decreases as  $\lambda_1$  decreases.

Consideration below is on the result that the distribution of CR in the case of ammonia co-firing has minimum point at approximately 0.8 of  $\lambda_1$  (in contrast to that in single coal combustion within the similar range of  $\lambda_1$ ). It is reported that  $NO_x$  emissions in single coal combustion decreases as  $R_{Ls}$ increases, however once a certain value is exceeded, it turns to increase<sup>(3)</sup>. This is because, when  $\lambda_1$  is low, NO<sub>x</sub> generation is suppressed in the first-stage combustion region, however a large amount of UBC remains, therefore NOx is generated in the second-stage combustion region when this UBC undergoes combustion. In other words, there is an optimal  $\lambda_1$ that minimizes the total-amount of NO<sub>x</sub> generation due to the fuel NO<sub>x</sub> generation in the first-stage combustion and combustion of UBC in the second-stage combustion. Volatile matter in coal and ammonia, which are both gaseous fuel, combust during first-stage combustion, however in the case of ammonia co-firing, more gaseous fuel is burned compared to single coal combustion. Therefore, it is more likely that UBC increases and the optimal  $\lambda_1$  becomes larger than that at single coal combustion.

NH<sub>3</sub> oxidation pathway in **Fig. 9** shows that NH<sub>3</sub> generates radicals of NH<sub>*i*</sub> (i = 0, 1, 2) due to the oxidation by OH and H, then NH<sub>*i*</sub> reacts with O, H, or OH to create NO, and



Fig. 9 NH<sub>3</sub> oxidation pathway<sup>(1)</sup>

meanwhile  $N_2$  is generated through a reaction between NO and  $NH_i^{(1)}$ . In the first-stage combustion region where air ratio is relatively low, there is not an abundance of O and OH, therefore an NO reduction reaction caused by  $NH_i$  is superior. It is presumed to be the reason why, even when co-firing with ammonia high in nitrogen content,  $NO_x$  generation can be suppressed to the same level as single coal combustion.

Moreover, the increase in *CR* in relation to  $\lambda_1$  increase is relatively smaller for ammonia co-firing. That means that ammonia co-firing will be much more effective in suppressing NO<sub>x</sub> generation than single coal combustion in the higher  $\lambda_1$  conditions, or in other words, when the twostage combustion conditions are low. There is a strong tendency for sulfidation corrosion to occur in a reducing atmosphere, however by applying pulverized coal/ammonia co-firing, the possibility of reducing sulfidation corrosion of the heat transfer tube near the burner while suppressing NO<sub>x</sub> generation is suggested.

#### 3.3 Effect of heat input on flue gas composition

When applied to actual boilers, it is expected that ammonia co-firing should be applicable at wide load condition. In the low load range, there is concern about combustion stability and emission characteristics of the ammonia co-firing because the temperature in the boiler furnace decreases. In this section, in order to confirm whether the flue gas composition is the same for ammonia co-firing and single coal combustion even in the low load range, pulverized coal and ammonia were co-fired at a low load and the flue gas composition was evaluated. Figures 10-(a) and -(b) show the relationship between heat input and NO<sub>x</sub> concentration in flue gas and the relationship between heat input and CR, respectively. Figure 11 shows the relationship between heat input and the UBC in the ash. Generally, in boiler furnaces, the  $\lambda$  increases in accordance with the decrease of load to ensure stable combustion. In this test, the situation was simulated by making the  $\lambda$  of 1.2 with a heat input of



Fig. 10 NO<sub>x</sub> concentration and *CR* as a function of heat input. In both cases of single coal combustion and ammonia co-firing, *CR* decreases with the increase of heat input.



Fig. 11 UBC as a function of heat input. At a heat input of 7.3 MW, UBC of ammonia co-firing is about twice as much as that of single coal combustion.

8.6 MW, however with a heat input of 7.3 MW and 6 MW, the  $\lambda$  increased to 1.4 and 1.7, respectively.

Figure 10-(a) shows that NO<sub>x</sub> increases as heat input decreases in both single coal combustion and ammonia cofiring, however the increasing trend in ammonia co-firing is almost the same as that in single coal combustion. This suggests that even in the low load range, NO<sub>x</sub> in ammonia co-firing is almost the same as that in single coal combustion. From Fig. 11, however, it can be seen that with a heat input of 7.3 MW, the amount of UBC in the ash in ammonia cofiring is approximately twice that in single coal combustion. Under the conditions of the lower heat input, the ammonia injection velocity decreases. The penetration of ammonia in the burner is reduced, and the ammonia becomes easier to mix with the combustion air in the vicinity. As a result, the mixing of pulverized coal and combustion air was suppressed, which may cause an increase in the UBC in the ash. If ammonia is injected at the base flow velocity under these conditions, it may be possible to suppress the increase in UBC in the ash.

# 3.4 Effect of coal fuel ratio and nitrogen content on flue gas composition

In a commercial plant, subbituminous coal, which has a lower fuel ratio, may be used as fuel in addition to bituminous coal. In order to investigate the effect of the coal fuel ratio on the flue gas composition at ammonia co-firing, a combustion test with the coal of the lower fuel ratio was conducted. As shown in **Table 1**, the test coal had fuel ratios (FR) of 1.16 and 1.70. The heat input condition was fixed to 7 MW.

**Figure 12** shows the relationship between the ratio of FR and FN (coal nitrogen content shown in **Table 1**) and *CR* in the single coal combustion and ammonia co-firing conditions. Within the range of FR/FN in this test, however, *CR* is considered not to make significant difference as it is 11 to 13%. It was confirmed that the *CR* increases parallel to increases in FR/FN<sup>(4)</sup>. For the ammonia co-firing condition, it was about 0.4 and its change with FR/FN was small. The



Fig. 12 *CR* as a function of FR/FN. In both cases of single coal combustion and ammonia co-firing, *CR* has no significant change with the change of FR/FN.

result shows that the effect of coal FR/FN on NO<sub>x</sub> emissions at ammonia co-firing is small. This is presumably because 90% or more of the nitrogen contained in the injected fuel (coal and ammonia) comes from ammonia in the case of 20% ammonia co-firing and the absolute amount of the injected nitrogen does not significantly differ within the range in this test. In addition, this result suggests that ammonia does not affect NO generation at the test furnace exit in the case of ammonia co-firing, so that ammonia is considered to be reduced to N<sub>2</sub> on the way to the furnace exit even if it oxidizes to NO in the furnace. It is important to note that further data is required to investigate this consideration.

**Figures 13-(a)** and **-(b)** show the measurement values at the furnace exit of  $N_2O$  concentration and  $NH_3$  concentration, respectively. Since  $N_2O$  has a global warming coefficient 298 times greater than  $CO_2$ , it is not possible to expect a reduction in greenhouse effect when more  $N_2O$  is discharged at ammonia co-firing even if  $CO_2$  is reduced.  $N_2O$  concentration was measured to understand the value in the case of ammonia co-firing.  $NH_3$  concentration was measured to check if ammonia injected as fuel reliably burned.

 $N_2O$  concentration was 5 ppm at single coal combustion in the test conditions. In the case of ammonia co-firing,  $N_2O$ concentration was 6 to 10 ppm, which is 1 to 5 ppm higher compared to single coal combustion. This is because the furnace internal temperature is lower in ammonia co-firing than in single coal combustion. Flame radiation of ammonia co-firing is lower than that of the single coal combustion due to smaller amount of the coal particle and the lower flame temperature. The value in ammonia co-firing was larger than that in single coal combustion presumably because  $N_2O$  in single coal combustion is more likely to be generated with lower temperature<sup>(2)</sup>. Note that a large coal fired boiler has larger heat input than this test equipment. With the larger heat input,  $N_2O$  emissions in ammonia co-firing are considered to be the same level as that in single coal



Fig. 13  $N_2O$  and  $NH_3$  emission with different FR coals.  $NH_3$  concentration of ammonia co-firing is the same level as that of single coal combustion.

combustion because the temperature in furnace is sufficiently high even in ammonia co-firing.

 $NH_3$  concentration was 1 ppm at single coal combustion and 0.5 ppm or less in the ammonia co-firing conditions, which is the same level as single coal combustion. Therefore, it is considered that ammonia injected as fuel is almost completely burned in the furnace in the range of the tested coal (FR 1.16 to 1.70).

### 4. Conclusion

With the ammonia co-firing burner developed in this study, combustion tests were conducted to evaluate the effect of difference in heat input and coal fuel ratio on the flue gas composition at the ammonia co-firing ratio of 20%. The results are as follows:

- (1) In ammonia co-firing at rated load, NO<sub>x</sub> concentration can be controlled at the similar level to that in single coal combustion by appropriately setting the ammonia injection velocity and  $R_{t,s}$ . In addition, it was confirmed that NO<sub>x</sub> concentration had the distribution with the minimum point according to the change of  $R_{t,s}$ .
- (2) Even with a heat input lower than the rated value, the NO<sub>x</sub> concentration in ammonia co-firing was the same level as that in single coal combustion.
- (3) With a coal fuel ratio and nitrogen content ratio (FR/ FN) between  $1.1 \times 10^2$  and  $1.25 \times 10^2$ , there was no significant difference in the *CR* in ammonia co-firing, therefore coal volatile matter and nitrogen content have little impact on NO generation. Moreover, NH<sub>3</sub> concentrations at the furnace exit were in the same range both in ammonia co-firing and in single coal combustion. Within the range of fuel ratio in this test, ammonia co-firing is suggested to be applicable because ammonia is considered to be almost completely consumed in the furnace.

In the case of low-load ammonia co-firing, the amount of UBC in the ash exceeded that of single coal combustion. This is considered a phenomenon unique to the test furnace

with a single burner. We will ascertain the characteristics in the combustion test with a multi-burner furnace in future. Moreover, in order to improve the versatility of ammonia cofiring technology, we will develop ammonia co-firing technologies for higher co-firing ratio.

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